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CFD mesh topology study of a truss geometry under forced oscillations

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#### CFD MESH TOPOLOGY STUDY OF A TRUSS GEOMETRY UNDER FORCED OSCILLATIONS

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### Introduction

- Deep oil and gas fields
  - Usually, Floating production units are more cost-efficient
    - Large subsea equipment employed



Åsgard oil field (Simmons, 2017)



### Introduction



• Equipment hydrodynamics ( $C_a$  and  $C_d$ )

Subsea equipment deployment (Wadhwa and Thiagarajan, 2009)

## Objective

This paper is a sequence of a previous works, it was done in order to progressively increase the complexity of the numerical model, this time with the focus of studying mesh topology and refinement

### Previous works

- Gomes et al., 2019 and Gomes et al., 2020
  - Simulations over disks and flat plates
- Morison equation
  - $-F = \frac{1}{2}C_d S |V|V + \rho C_M \nabla \dot{V}$ 
    - $C_M = 1 + C_a$
- Hydrodynamic coefficients
  - Chreim *et al.,* 2020
- Main dimensionless parameter:

$$- KC = \frac{U_M T}{D} = \frac{2\pi A}{D}$$



Experimental apparatus from (Tian et al., 2017) (left) and disk motion (right)

#### **Previous works**

- Commercial CFD Software
  - ANSYS Fluent with double precision
- Incompressible fluid
- Reference frame motion in the entire domain
  - $\zeta(t) = A\cos(\omega t)$
- 2d axisymmetric mesh and 3d unstructured mesh
- Unsteady state
- Viscosity model:
  - k-k<sub>l</sub>-ω



#### 2d mesh used in the previous work

#### Previous works

• Hydrodynamic coefficient results



## Current work

- Initially, the same approach used in the previous works was employed, but using a more complex geometry
  - Similar to a flat plate, but with increased porosity

- Later, due to the obtained results and the solution verification analysis, two new mesh topolgies are studied
  - In order to improve its outcome (hydrodynamic coefficients), reduce mesh uncertainty and computational cost
  - The study was based on the vorticity field results

Study case

## Study case

• Dimensions:

•  $(L_x = 465; L_y = 315; L_z = 15)[mm, mm, mm]$ 

- Motion characteristics:
  - $0,15 \leq KC \leq 3,00$
  - 10 values
- Domain dimensions:
  - $[-7,5 \times L_x; 7,5 \times L_x][mm]$
  - $\left[0; 7, 5 \times L_y\right][mm]$
  - $\left[-7,5 \times \max\left(L_x; L_y\right); 7,5 \times \max\left(L_x; L_y\right);\right][mm]$
- Symmetry plane
- Mesh topology -> Poly-hexcore



## Study case - Domain and mesh



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KC = 0,75





KC = 1,50





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KC = 3,00



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• Drag coefficient results



## Study case - Solution verification

- The solution verification analysis was based on the work of Eça and Hoekstra (2014) and was performed using the Numerical Uncertainty Analysis (NUA);
- The adopted method to estimate the uncertainty,  $U_{\phi}$ , uses an integral or local result from the simulation, and requires a series of geometrically similar meshes with a systematic refinement between each one of them;
- Due to the computational effort required only the value of KC = 1,00 was simulated over 6 different meshes;
- In this study only the spatial discretization was analyzed.



|                                       | VVM2-       | VVM-  | VVM0        | VVM+   | VVM2+      | VVM3+   |
|---------------------------------------|-------------|-------|-------------|--------|------------|---------|
| Relative step size                    | $4\sqrt{2}$ | 4     | $2\sqrt{2}$ | 2      | $\sqrt{2}$ | 1       |
| Number of elements (10 <sup>3</sup> ) | 205,8       | 472,2 | 1204,2      | 3001,9 | 7819,4     | 20786,4 |

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### Study case - Solution verification results



- Investigation of the regions of major interest to be further refined
  - Creation of new mesh topologies that can better capture both the forces generated by the movement and the resulting vortices
- Flow results from the previous simulations were analyzed (KC = 1,00), based on the vorticity field in a Y-plane that passes through the structure
- It was established an absolute vorticity value of 5 s<sup>-1</sup> as basis for comparison between the simulations
- Only the first 3 squared bars of the structure, from left to right, were selected and numbered in the same order



#### VVM2-



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#### VVM0







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Units in [mm]

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|        | Vertical direction |       |                  |       |       |                  | Horizontal direction |       |                  |       |       |       |
|--------|--------------------|-------|------------------|-------|-------|------------------|----------------------|-------|------------------|-------|-------|-------|
|        | Maximum Velocity   |       | Minimum Velocity |       |       | Maximum Velocity |                      |       | Minimum Velocity |       |       |       |
|        | Bar 1              | Bar 2 | Bar 3            | Bar 1 | Bar 2 | Bar 3            | Bar 1                | Bar 2 | Bar 3            | Bar 1 | Bar 2 | Bar 3 |
| VVM 2- | 40,00              | 32,00 | 40,00            | 32,00 | 8,00  | 4,00             | 16,00                | 16,00 | 16,00            | 16,00 | 16,00 | 16,00 |
| VVM-   | 50,91              | 28,28 | 39,60            | 33,94 | 16,97 | 19,80            | 16,97                | 16,97 | 11,31            | 16,97 | 16,97 | 11,31 |
| VVM0   | 36,00              | 28,00 | 40,00            | 20,00 | 10,00 | 14,00            | 12,00                | 16,00 | 12,00            | 16,00 | 16,00 | 12,00 |
| VVM+   | 28,28              | 25,46 | 33,94            | 16,97 | 8,49  | 14,14            | 8,49                 | 11,31 | 8,49             | 14,14 | 14,14 | 11,31 |
| VVM2+  | 30,00              | 26,00 | 32,00            | 18,00 | 10,00 | 16,00            | 8,00                 | 10,00 | 8,00             | 16,00 | 16,00 | 14,00 |
| VVM3+  | 26,87              | 28,28 | 29,70            | 22,63 | 9,90  | 16,97            | 7,07                 | 8,49  | 7,07             | 18,36 | 14,14 | 14,14 |

## New mesh topologies

- Two new refinement bodies
- Refinement B (Orange)
  - The same used in domain 1
- Refinement C (Green)
  - Common to both: minor radius of 37,5 mm; major radius of 62,5 mm
- Refinement D (Blue)
  - Domain 2: radius of 18,75 mm
  - Domain 3: minor radius of 27,5 mm; major radius of 47,5 mm

#### Domain 2



Domain 3



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## New mesh topologies

- Two new refinement bodies
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Domain 3



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#### New mesh topologies



| Domain    | Max. Dim.<br>[mm] | Ref. A [mm] | Ref. B [mm] | Ref. C [mm] | Ref. D [mm] | Number of<br>elements<br>(10 <sup>6</sup> ) |
|-----------|-------------------|-------------|-------------|-------------|-------------|---|
| VVM0 (1)  | 80                | 8           | 4           | -           | -           | 1,20  |
| VVM3+ (1) | 20√2              | 2√2         | $\sqrt{2}$  | -           | -           | 20,79                                       |
| 2         | 80                | 8           | 4           | 2           | 1           | 5,21  |
| 3         | 80                | 8           | 4           | 2           | 1           | 9,58  |

# Mesh topology study – Solution verification

 Due to the computational effort required only the value of KC = 1,00 was

simulated only over 3

different meshes



Domain 2



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## Mesh topology study – Mesh efficiency

• Comparison between all simulated meshes for KC = 1,00



### Conclusion

- This paper presented a study related to mesh topology and refinement of CFD simulations;
- It is a continuation of a research related to the calculation of hydrodynamic coefficients of subsea structures under sinusoidal motion using CFD simulations;
- In the new mesh topologies there was a reduction of both computational cost and numerical uncertainty, with a similar final coefficient result;
- If more focus is given to the value of the obtained coefficients, domain 2 may be a better choice, because of its analogous results and lower computational cost; Whereas, if more focus is given to the resultant flow topology, domain 3 may be recommendable, since it has a higher number of elements with small size near the geometry.

#### Conclusion

- In future stages, it is intended to estimate the uncertainty related to both spatial and temporal discretization of the simulations with the three different domains, in order to have a better picture of its most relevant uncertainties;
- It will be interesting to simulate using the presented method, more realistic geometries, which are representative of equipment used in the offshore industry.

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